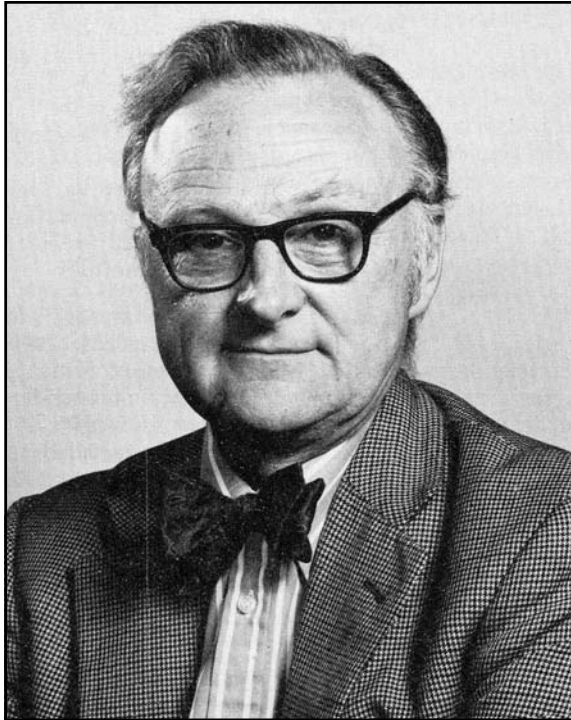


The Role of Fire Science

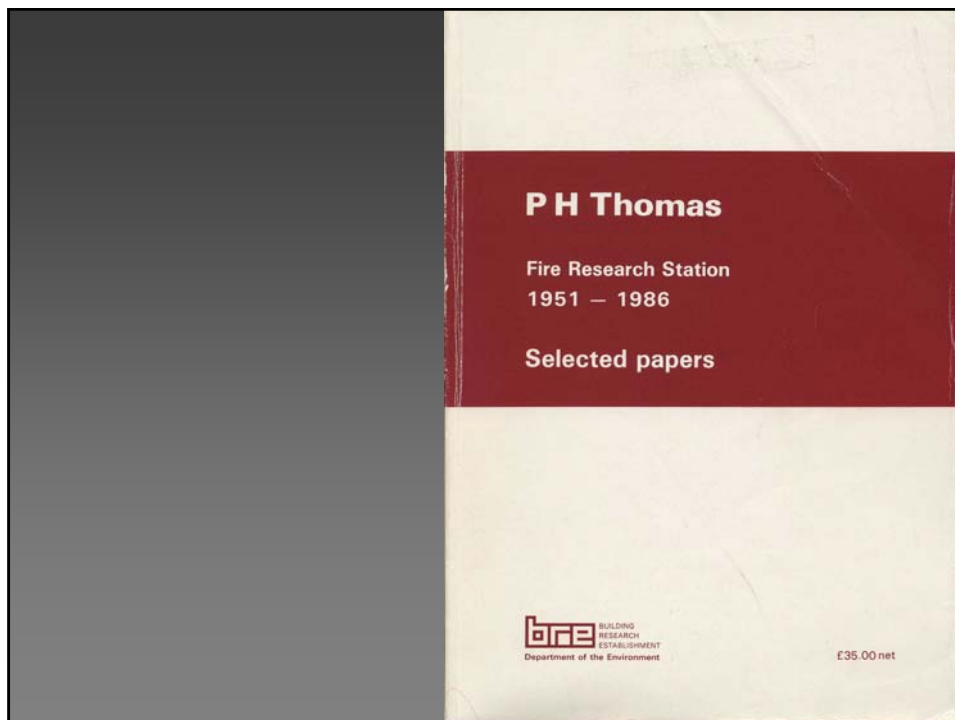
2008 Philip Thomas Lecture



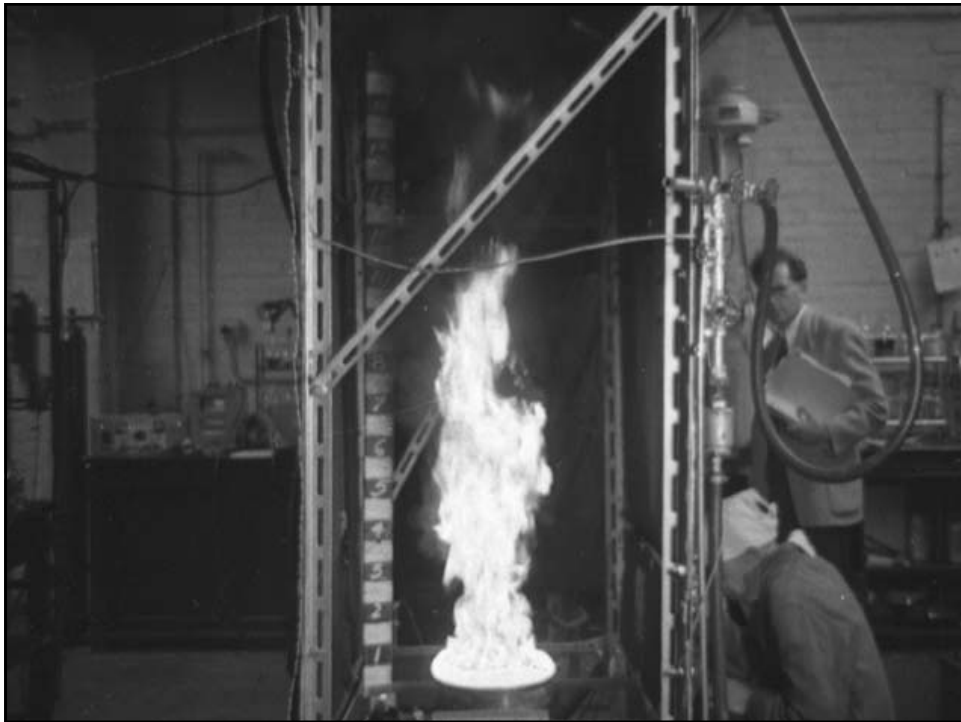
- Coordinator CIB W14, 1974-1994
- Chairman ISO TC92, 1976-1995
- First Chair IAFSS, 1985-1991

Seminal contributions to so many areas of the science of fire

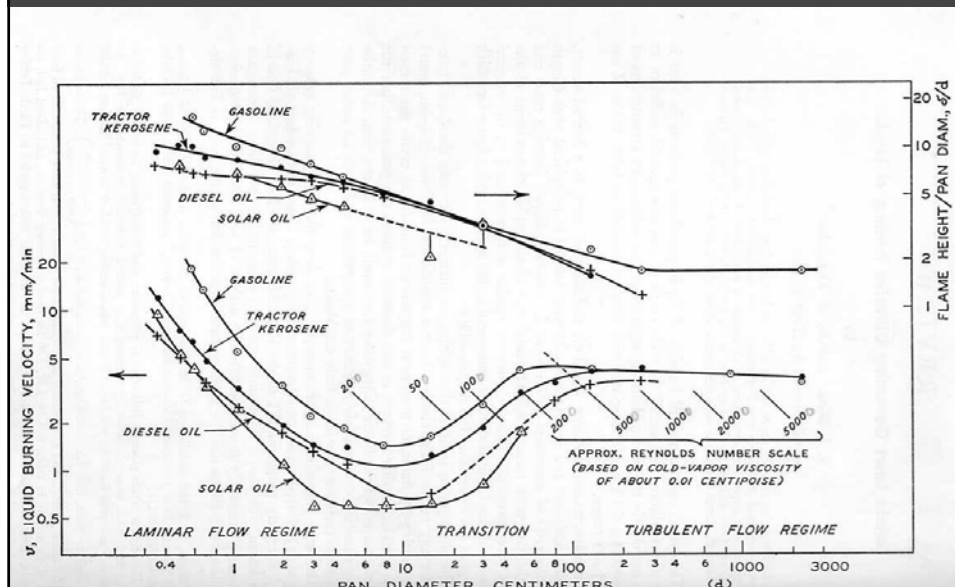
- self-heating
- thermal explosion
- ignition and extinction
- water sprays
- flame size
- air entrainment
- enclosure fire dynamics
- flashover
- flame spread
- smoke venting
- forest fires
- etc, etc



Buoyant Diffusion Flames



Blinov & Khudiakov's data (as presented by Hottel, 1958)



Flame height-dimensional analysis

- Dimensional arguments suggest flame length:

$$\frac{L}{D} \propto f \left(\frac{Q^2}{gD^5} \right)$$

if average air velocity

$$\propto L^{1/2}$$

f determined from entraining surface area, then

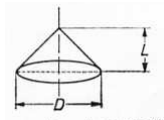
$$\frac{L}{D} \propto \left(\frac{Q^2}{D^5} \right)^{\frac{1}{2p+1}}$$

Flame height-dimensional analysis

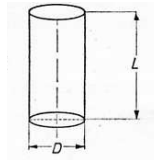
- where p is a shape factor, varying from 0 to 2 with increasing L/D

- $\frac{L}{D} \propto \left(\frac{Q^2}{D^5}\right)$; $\frac{L}{D} \propto \left(\frac{Q^2}{D^5}\right)^{\frac{1}{3}}$; $\frac{L}{D} \propto \left(\frac{Q^2}{D^5}\right)^{\frac{1}{5}}$

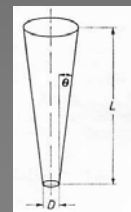
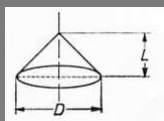
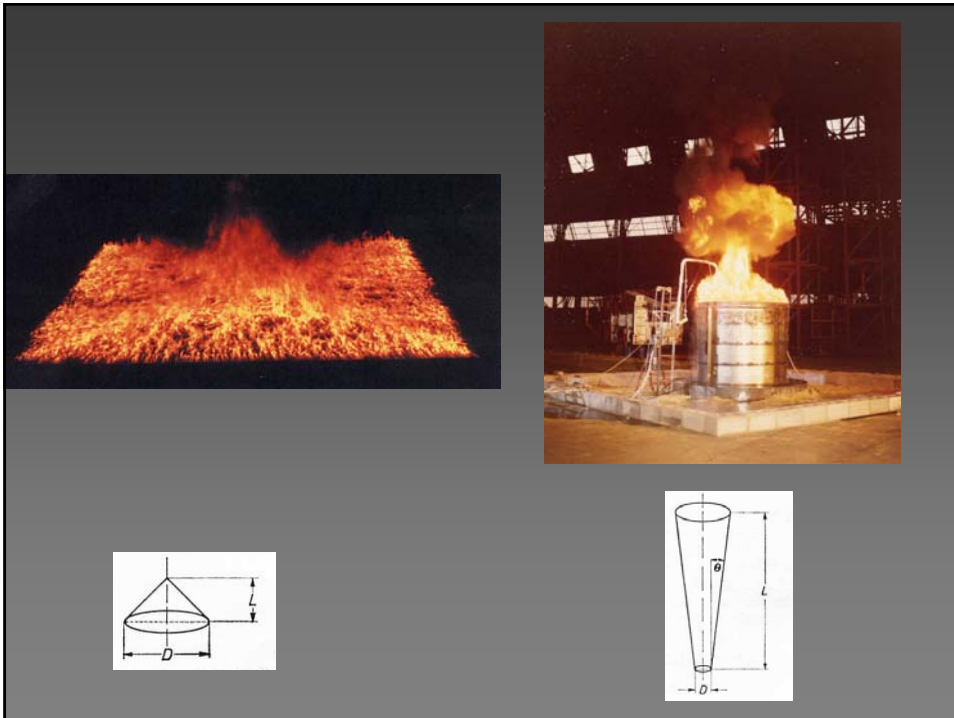
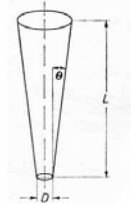
$p=0$



$p=1$



$p=2$



Flame height data

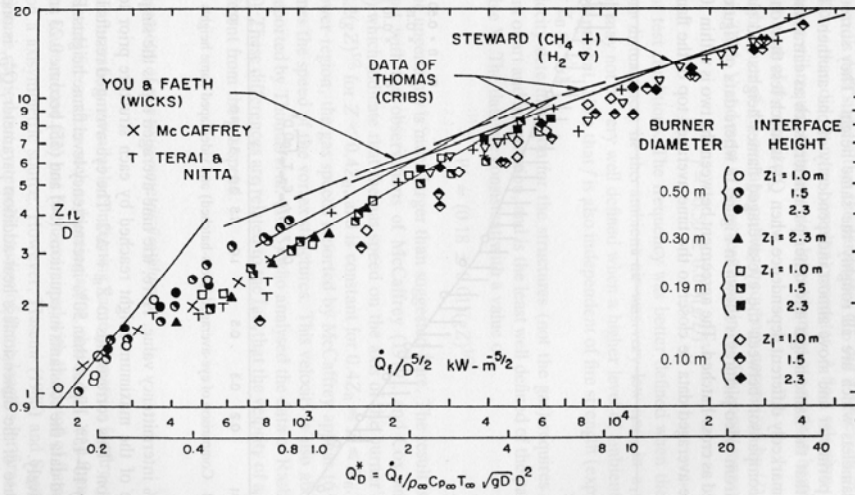


Figure 17 Eye-averaged flame-height data correlation.

Rasbash 1954



Model for flame velocity

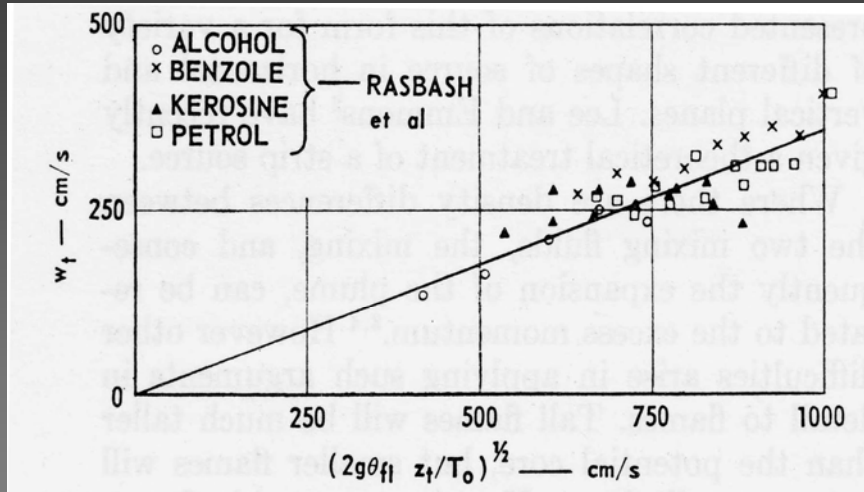
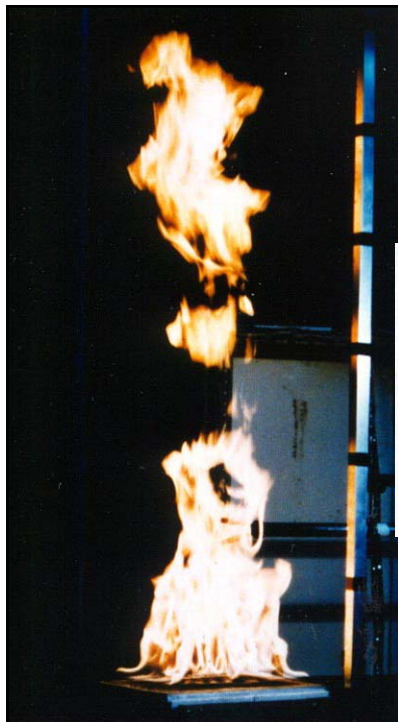
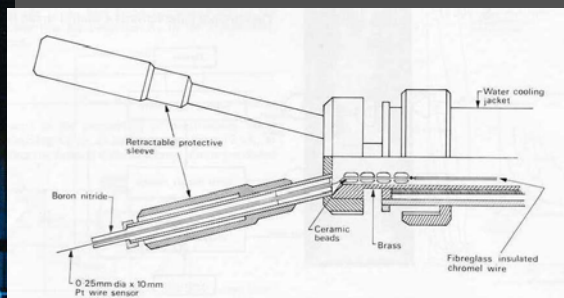


FIG. 1. Upward velocity of flame tip.



Transit time of natural fluctuations



Fluctuations in temperature and flame ionization

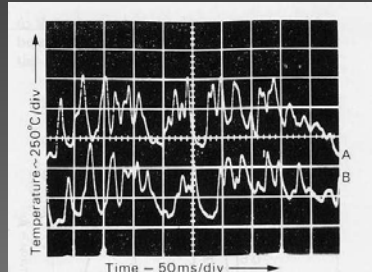


FIG. 2 - Temperature fluctuations obtained at two points A and B in fire.

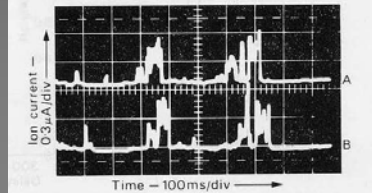
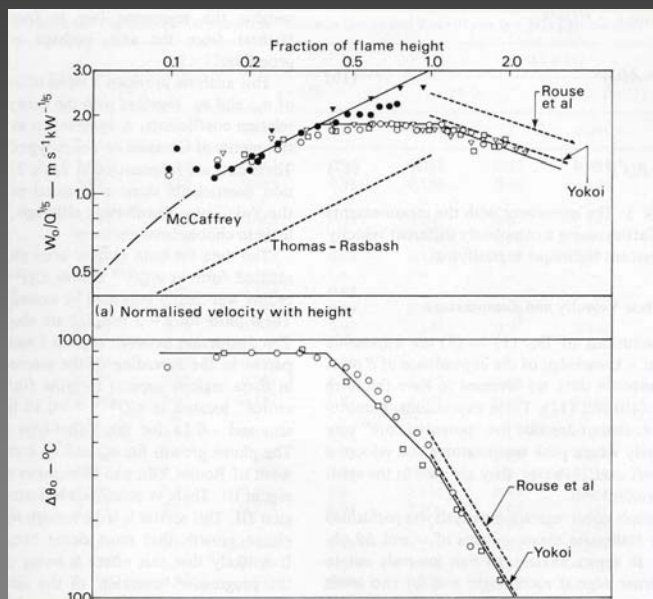
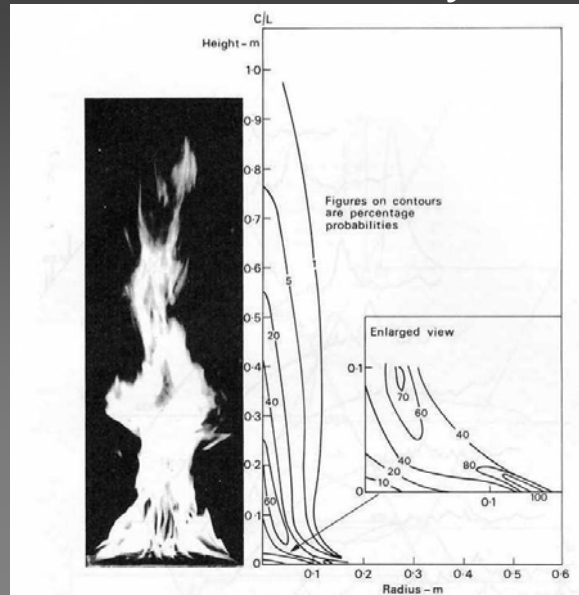


FIG. 3 - Ion current fluctuations at points A and B in fire.

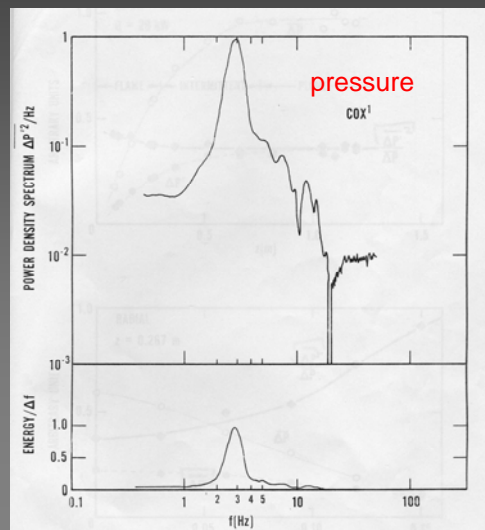
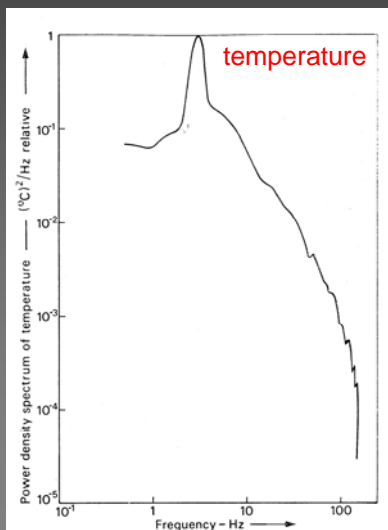
Centreline velocity & Temperature



Intermittency

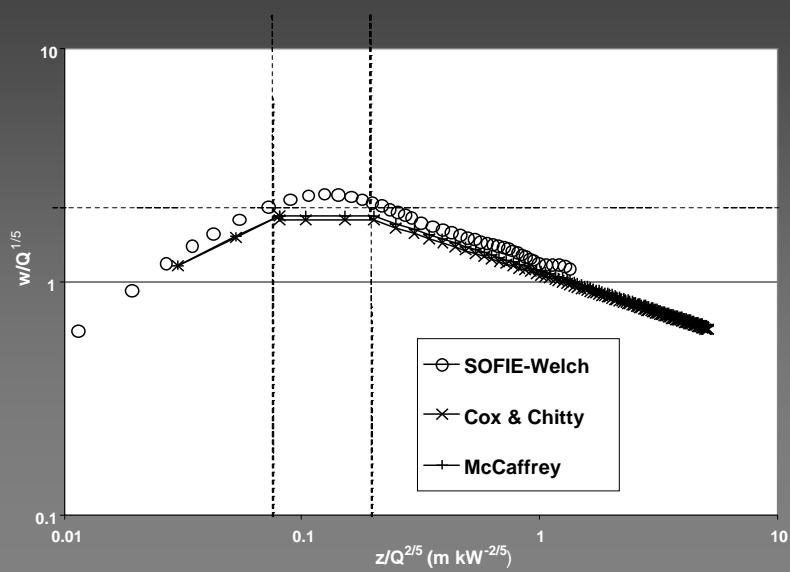


Spectra of gas temperature and pressure difference

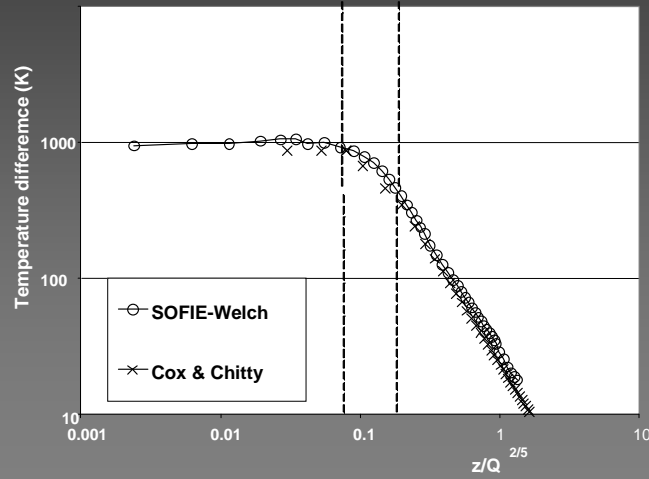


Fire Modelling

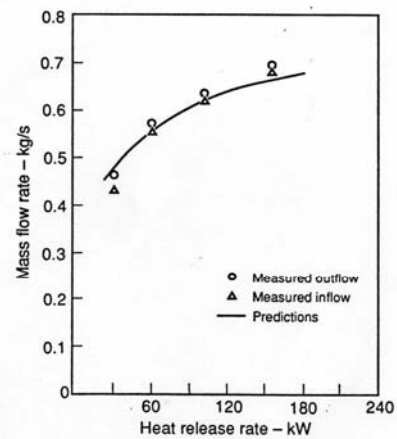
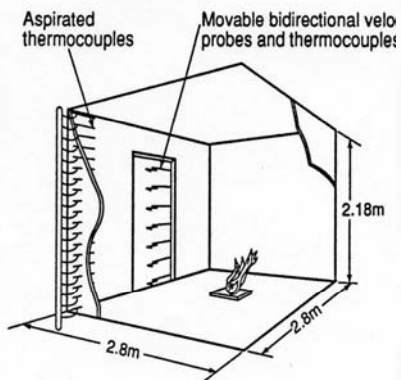
RANS CFD centreline velocity



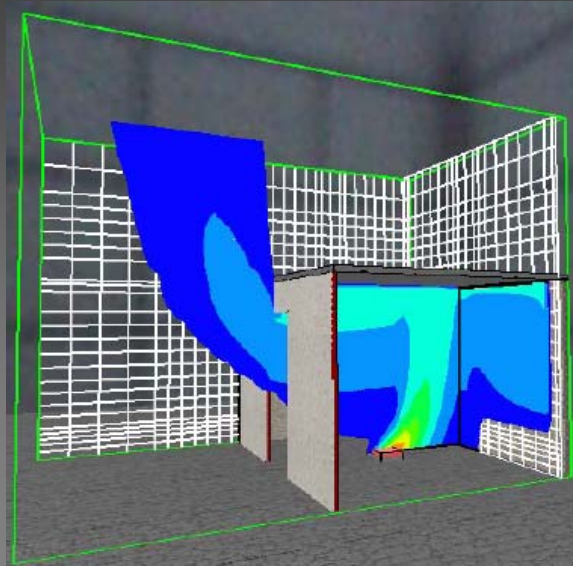
RANS CFD Centreline temperature



Early JASMINE RANS prediction (1983)



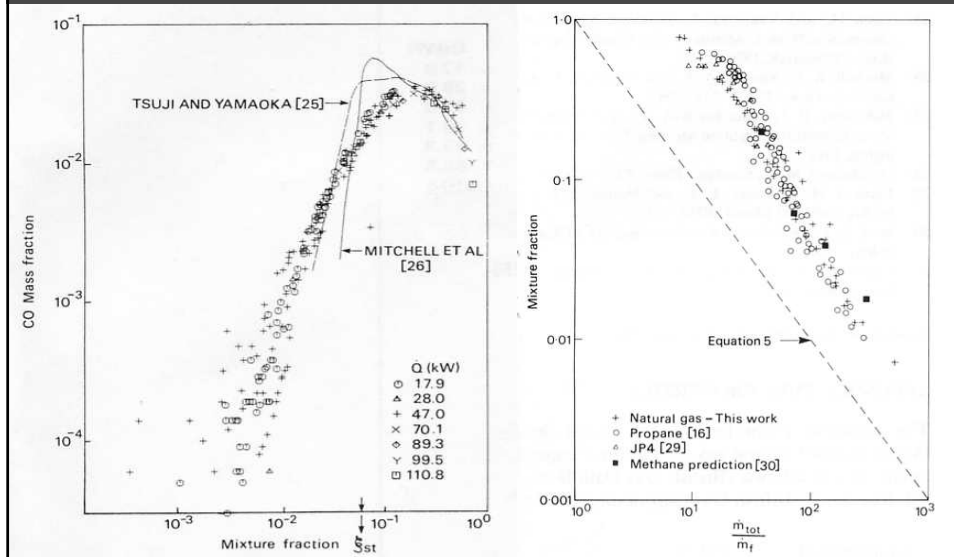
Plume deflection as predicted by CFD



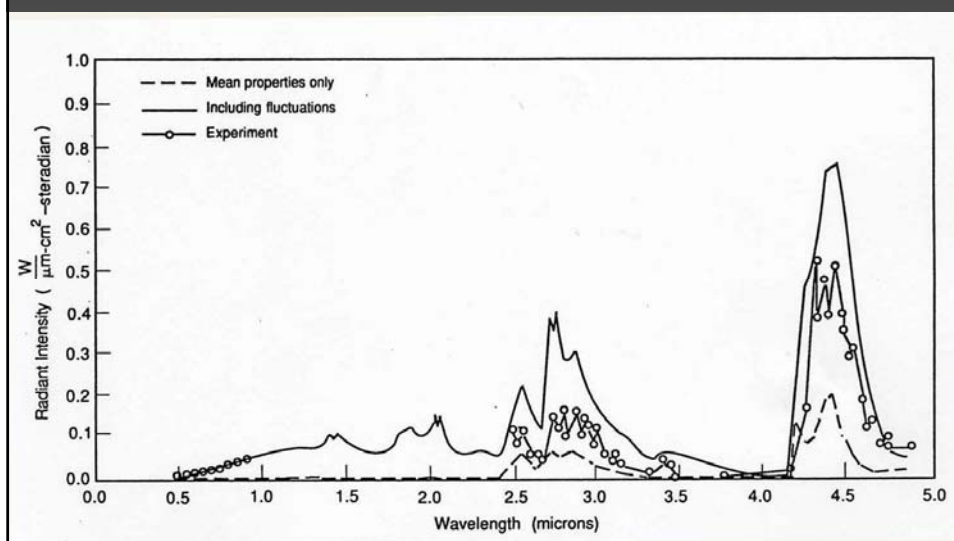
Development of “sub-models”

- Turbulence-chemistry interactions
- Turbulence-radiation
- Two phase flow (sprinkler droplets, fibre/particulate dispersion)
- Gas-solid interactions

Non-equilibrium species



Predicted spectral distribution using laminar flamelet model (Syed et al)



0.5 and 1.0mm
droplet
trajectories over
400kW fire
(Ayres)

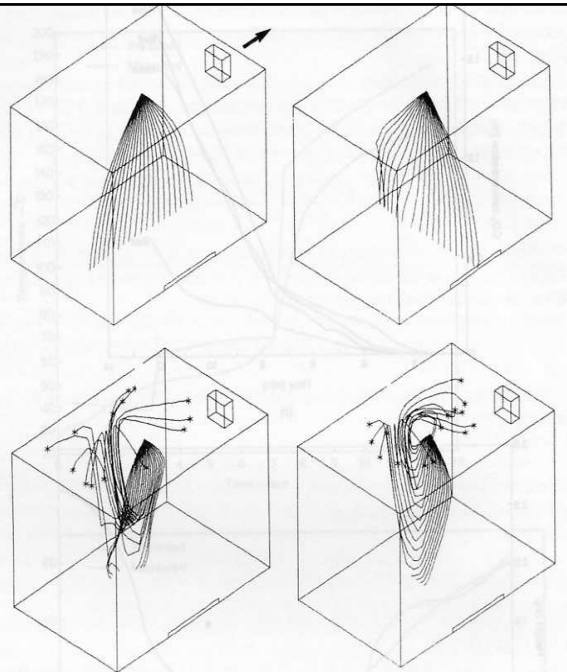
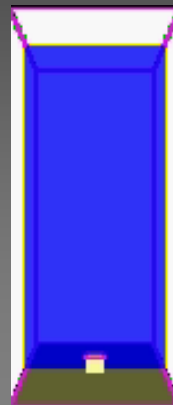


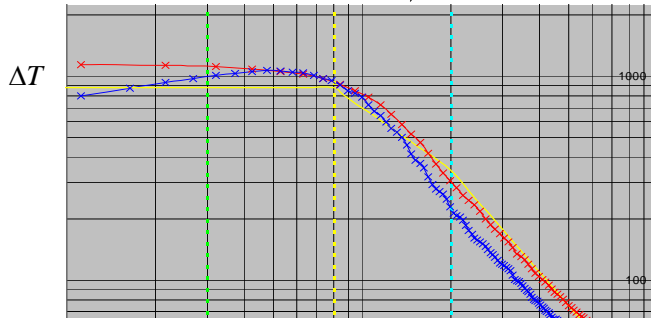
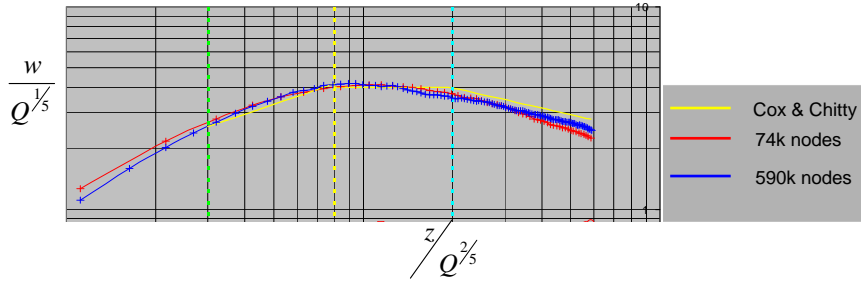
Figure 24 Predicted trajectories of water droplets injected over the 400kW fire in the Lawrence Livermore fire test cell (see Figure 22).

Large Eddy CFD

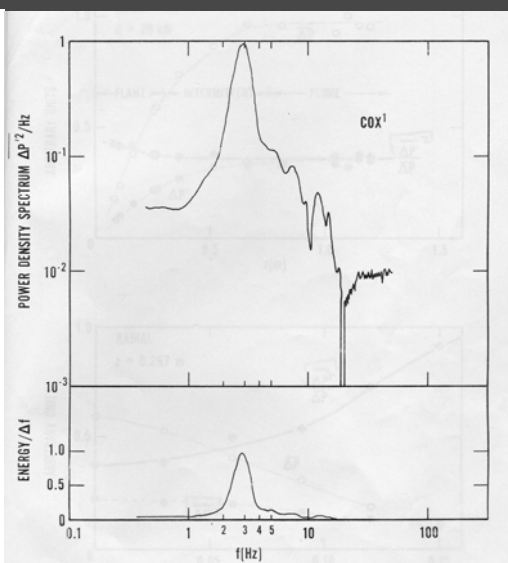
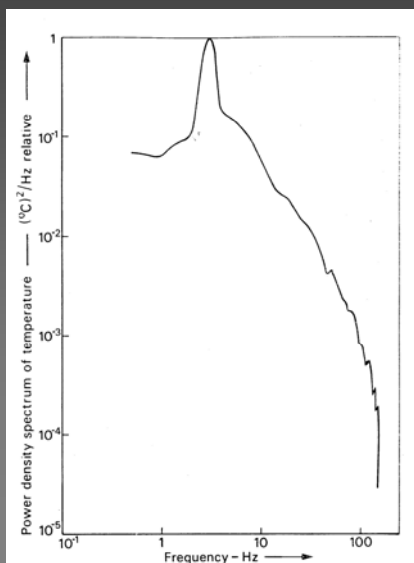
NIST FDS model



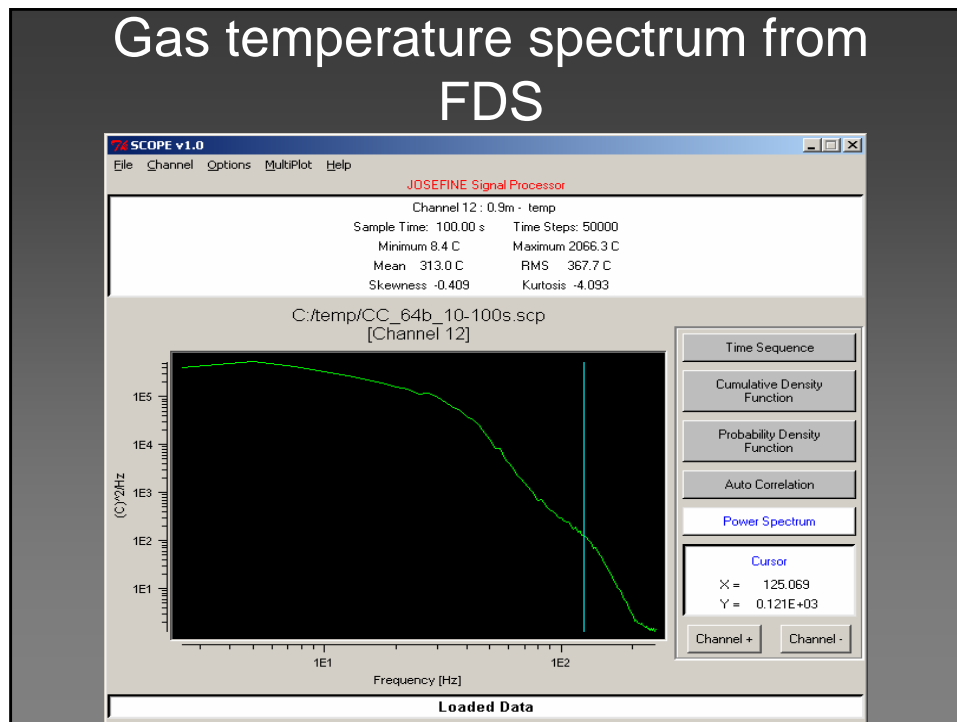
LES CFD predictions (FDS-Welch)



Spectra of gas temperature and pressure difference



Gas temperature spectrum from FDS



Research needs

- While CFD models capture the broad behaviour more research is needed to understand laminar-turbulent transition above source
- Rayleigh-Taylor instability causing toroidal vortex engulfment of air?
- Understanding turbulence-chemistry (CO formation) and turbulence-radiation interactions; boundary layer flow

FSE

- Developing maturity of fire engineering came at a time when political trend was favouring performance-based regulatory reform
- However it is not yet mature and practitioners need to be very cautious
- They need to understand the tools available and to know their limitations

An example of mis-use of RANS CFD

| Heat release rate /MWm ⁻¹ | | 1 | 2 | 3 | 4 | 5 | 1 | 2 | 3 | 4 | 5 |
|--------------------------------------|-------------------------|--|------|------|------|------|--|----|----|-----|-----|
| Case | Free boundary condition | Maximum temperature (excluding inside the heat source) in the hall /°C | | | | | Maximum temperature at the free boundary /°C | | | | |
| BF1 | FC1 | 580 | 1320 | 2320 | 3620 | 5160 | 53 | 76 | 96 | 113 | 130 |
| | FC2 | 568 | 1300 | 2290 | 3540 | 5050 | 50 | 71 | 90 | 108 | 125 |
| BC1 | FC1 | 565 | 1250 | 2260 | 3600 | 5270 | 61 | 69 | 88 | 107 | 125 |
| | FC2 | 563 | 1300 | 2330 | 3670 | 5350 | 51 | 73 | 93 | 113 | 132 |
| BC2 | FC1 | 630 | 1440 | 2690 | 4410 | 6220 | 57 | 66 | 84 | 102 | 120 |
| | FC2 | 615 | 1480 | 2750 | 4460 | 6720 | 50 | 70 | 89 | 108 | 126 |

Table 4: Maximum temperature predicted

Modelling-Problems eg RANS

Table 1
Comparison between models and with experiment

| | h_N/h_0 | Inflow (kg/s) | Outflow (kg/s) | Temperature (°C) |
|-------------------|-----------|---------------|----------------|------------------|
| VHS | 0.418 | 0.568 | 1.266 | 137 |
| Eddy breakup | 0.407 | 2.112 | 1.117 | 117 |
| PrePDF | 0.450 | 1.165 | 1.209 | 113 |
| Lewis et al. [21] | 0.546 | 0.521 | 0.523 | 128 |
| Experiment [20] | 0.561 | 0.554 | 0.571 | 129 |

Serious risk of CFD mis-use

- SFPE Handbook, Chapter on CFD, 2002
- HSE (Guidance for HSE Inspectors: Assessment of CFD), 2002
- ODPM-Computer Model Performance Assessment Scheme (CoMPAS), 2004

But determinism can only go so far....

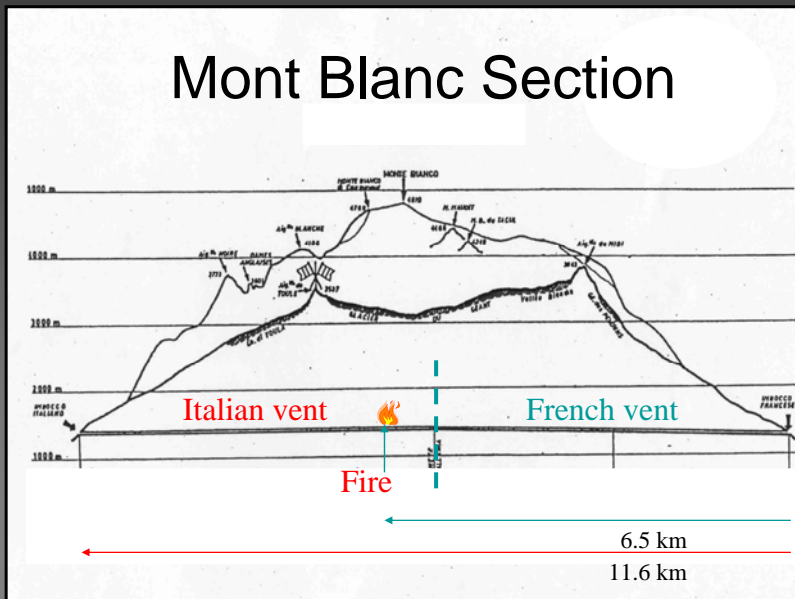
An illustration

Mont Blanc Tunnel Fire

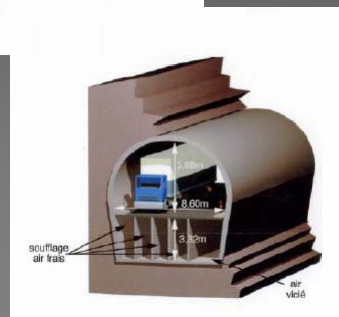
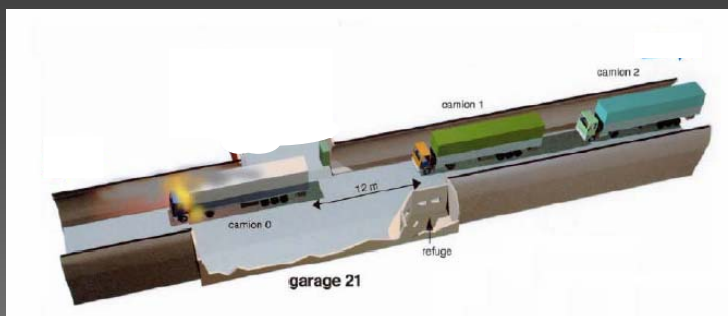
39 fatalities; March 1999



Mont Blanc Section



Tunnel Section



The Mont Blanc Tunnel Fire

- all deceased were on French side of the truck, 38 within 1km of fire-most still in their vehicles
- decisions on smoke ventilation were critical; responsibilities divided between two separate control rooms at French & Italian ends of tunnel
- French half extracted at roof apex; Italian side supplied air at roof apex

Tribunal 2005

We were asked to conduct:

CFD simulations of the gas phase conditions in the tunnel coupled with tenability and escape simulations for the occupants:
for

- the conditions of the tragedy
- and for possible alternative ventilation choices particularly the “Cas Consignes”

Modelling Philosophy

- there are things **we know** (dimensions; gradients of tunnel, construction materials used, dimensions location, contents of truck, ventilation settings)
- there are things **we don't** (fire "size" & growth rate, external pressure difference between ends of tunnel)
- assumptions/approx necessary

Modelling (CFD & Human Factors)

BRE JASMINE model

- transient calculations of whole length of tunnel modelled
- three gradients of tunnel included
- rate of fire growth estimated from truck fire load & available opacity meter data
- Smoke visibility & CO/HCN concentrations from engineering correlations

CFD simulations coupled to Human Factor modelling (FED & people movement)

Initial condition is simulation of cold flow before the fire

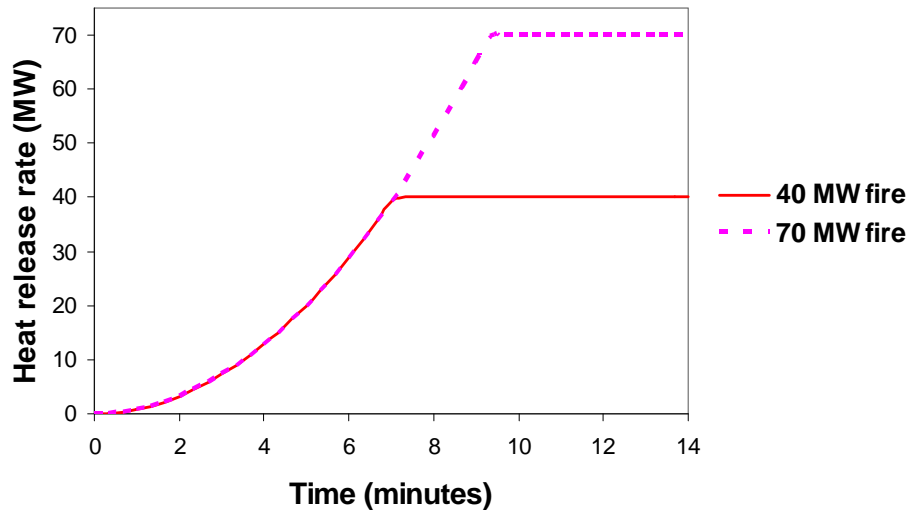
- steady-state “cold flow” modelled first; compared to anemometers in tunnel
- external pressure difference between tunnel ends based on the meteorological data
- transient fire source then added

Fire Modelling

Modelling detail of fire propagation through complex geometries of an assortment of different “fuels” is not possible (margarine, flour, tyres, kerosene, polyurethane sandwich board etc)

so assumptions/approximations need to be made

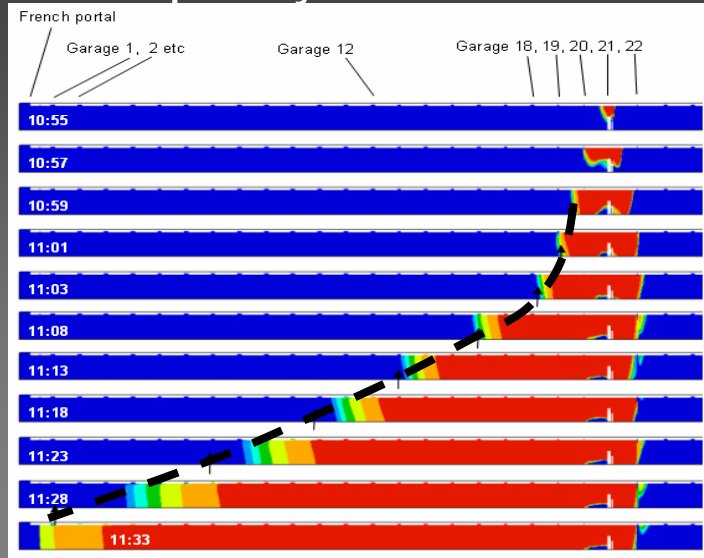
Assumed fire sources



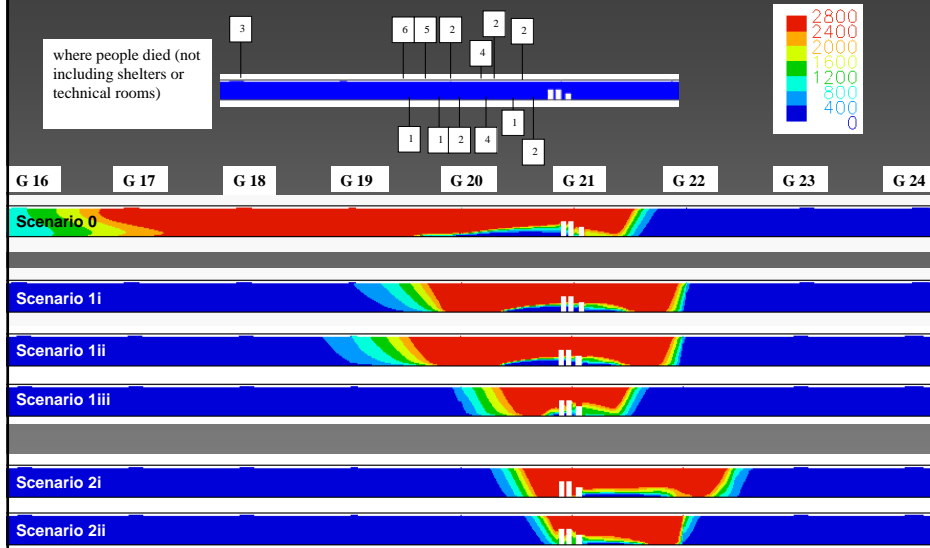
The Fire-assumptions for fuel

- mass release rate of fuel determined from the heat release rate curve
- heptane chemistry used as a “surrogate” fuel
- smoke yield of the actual materials was represented by rigid polyurethane, a typical “smokey” fuel

Visibility predictions and opacity meter data

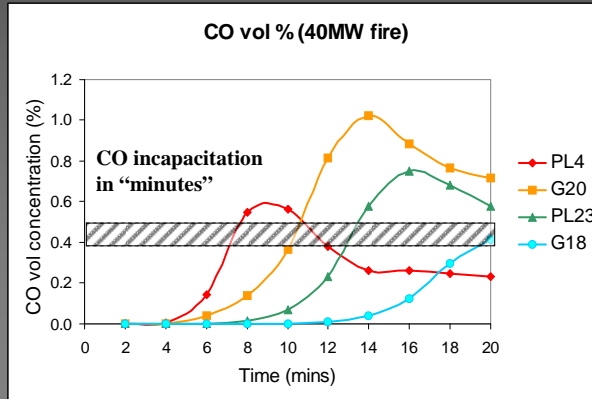


Carbon monoxide contours at 11:13 am

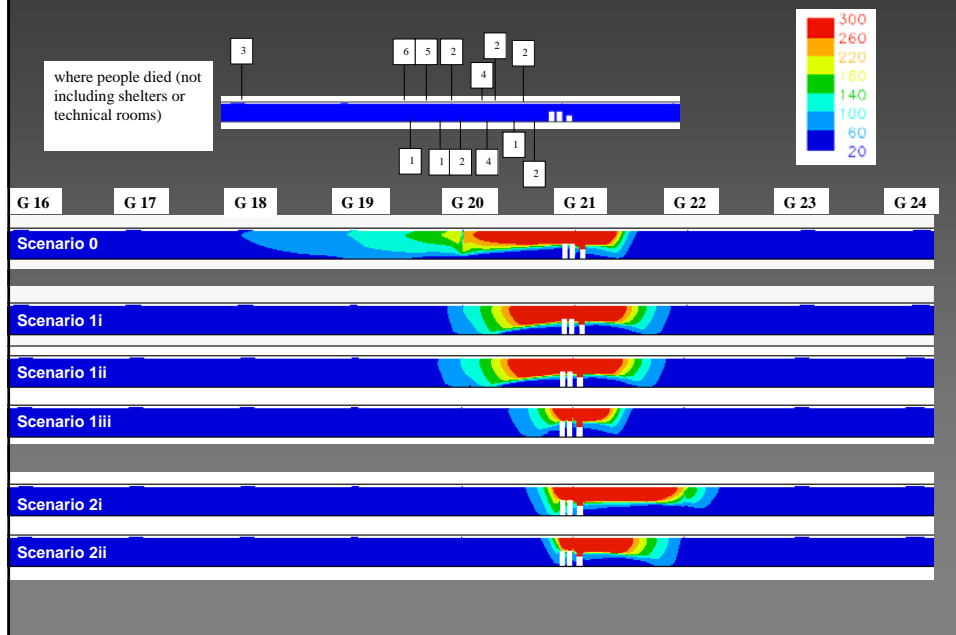


CO concentration at “head height”

- 2m above road surface



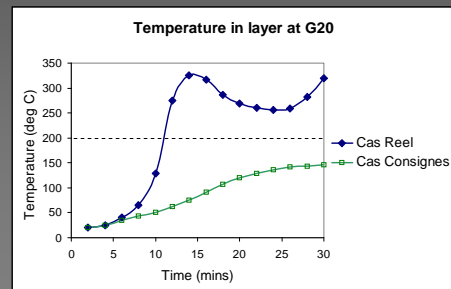
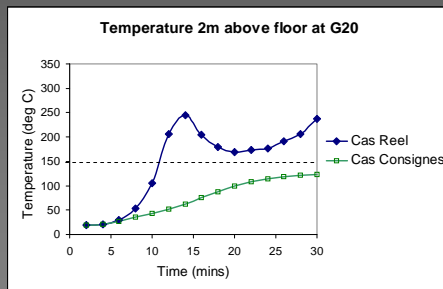
Gas temperature contours at 11:13 am



Gas temperature Garage 20, Scenarios 0 and 1(i)

150°C immersion temperature tolerate for “minutes”

200°C \cong 2.5kW/m²; pain on bare skin-1min



Model conclusions

- toxic gas hazards spread quickly to G18 (900m) by about 11.13 (20 minutes)-38 people died within 932 m of Truck, PL0
- thermal hazard develops rapidly spreading to G20 (300m) by about 11.04 (11 minutes)-19 people died within 303 m of PL0
- the detailed gas-phase conditions used as inputs for human tenability and egress modelling for victims

How robust are these conclusions?

To ensure that our conclusions are robust and prudent, we considered further how sensitive they are to different assumptions for

- maximum fire size
- larger external pressure difference from Italy to France

“A Worst Case”

- Tenability analysis of survivability for an assumed more severe case of 40 Pa, 70 MW fire concludes that
 - the occupants of VL20 (407 m from Truck, PL0) would have died at 540 m from PL0 rather than 932 m, where they were actually found
- Thus this “worst case” possibility is less compatible with the known facts on the day of the incident

Conclusion

- With the approximations made, the 10Pa 40 MW case best describes the disaster of 24 March 1999
- For Scenario 1, the Cas Consignes, with all ventilation prescriptions studied, the conditions suggest that some, possibly many, occupants could have survived.
- However, close to the Truck, PL0, the thermal hazard remains severe threatening any occupants who remain in their vehicles

What do we learn from this example?

- Example illustrates current strengths and weakness of our current knowledge
- Deterministic models are very powerful but
 - are not yet complete; there are many phenomena not yet fully understood
 - deterministic approaches can only go so far in the “real world”
- Probabilistic modelling requires greater attention

Probabilistic modelling in FIREGRID (Koo et al, 2008)

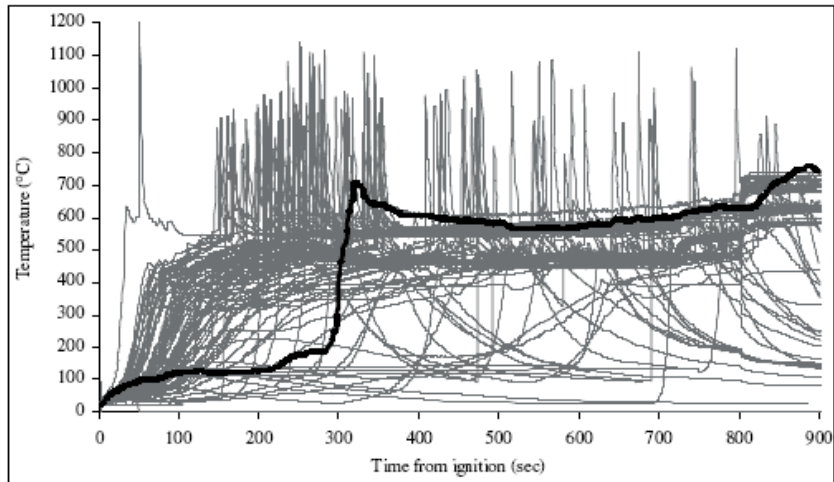
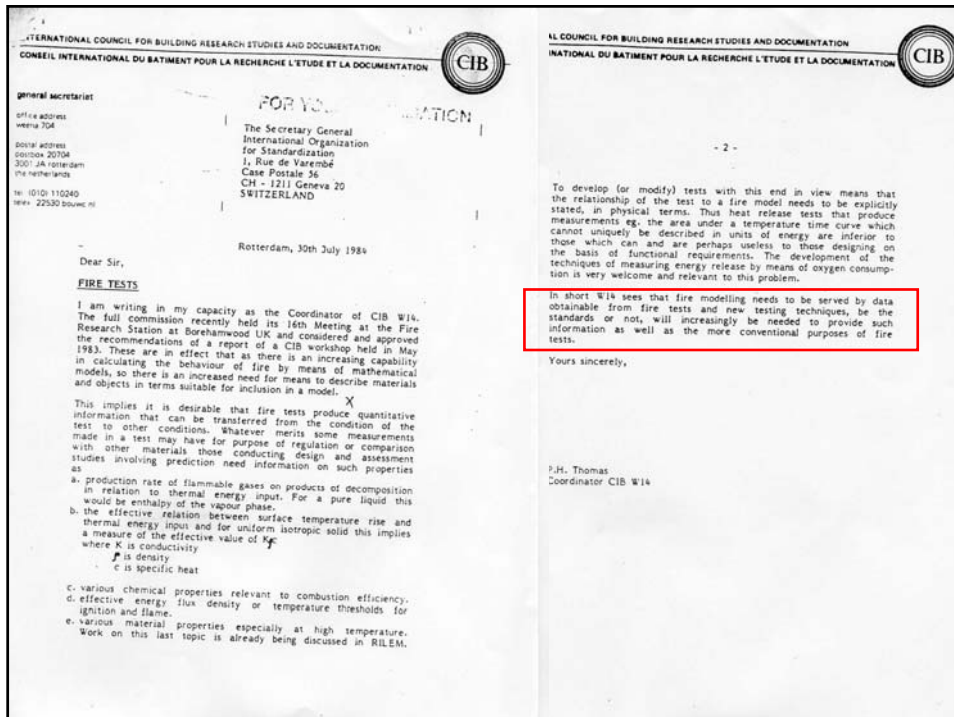


Fig. 5. 100 CRISP outputs among 2000 simulations.

Fire Testing & International Standards

Prof Kunio Kawagoe

“The purpose of fire research is to do away with the fire resistance test”



ISO TC92 new responsibility, 1995

“Fire Tests on Building Materials, Components and Structures”

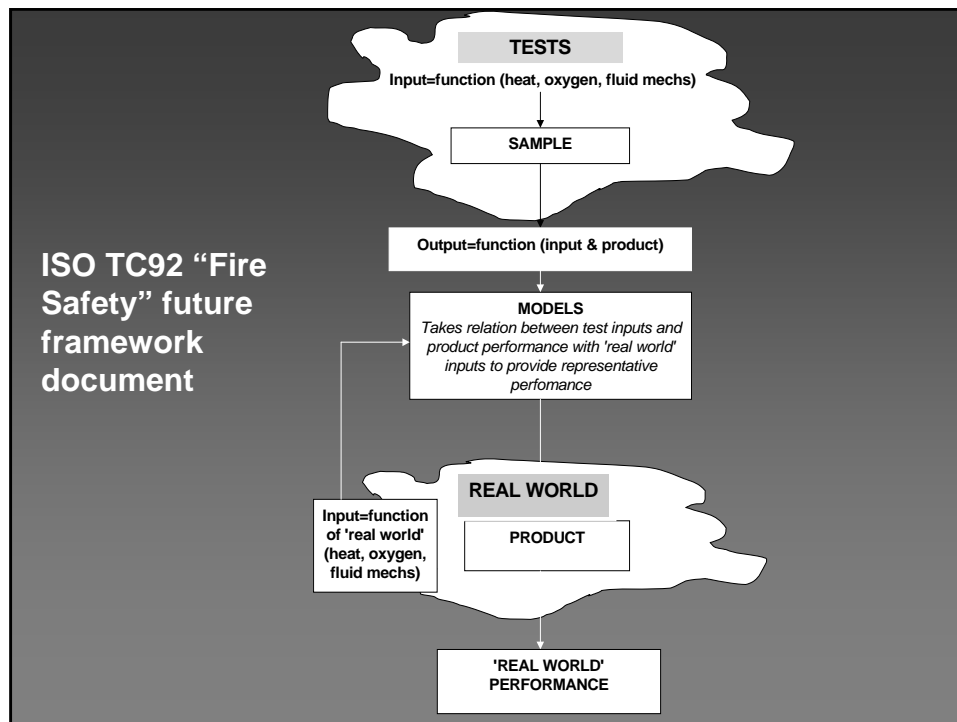
to

“Fire Safety”



ISO Fire Tests for FSE

- that 'product' performance in the test is provided in quantitative terms
- that exposure conditions must be provided in quantitative form
- that processes in the test are sufficiently well prescribed that they can be modelled
- that performance from the particular conditions of the test must be translatable by predictive methods to the design environment



“Unfortunate” political influences

- Single Burning Item test
- Privatisation of many national labs
- Lack of strategic research policy; reduction in funding for research

SBI Test

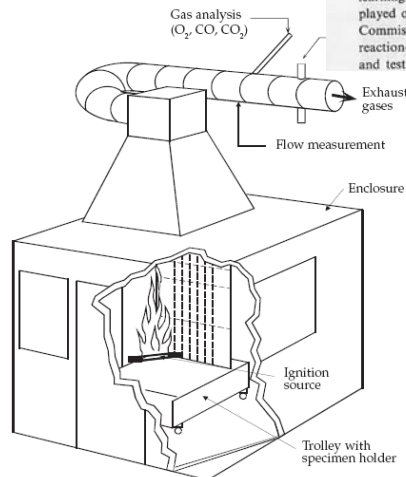


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0379-7112/97/\$17.00

Editorial 2

Redefining the value of π in the European Union

There is an apocryphal story of a 19th century American legislature which tried to define the value of π as 3.0, on the basis that this would help school children in learning mathematics. A similar tale of regulators trying to define science is now being played out in the EU, but regrettably it is not apocryphal. In 1994, the European Commission promulgated a decision that established a new scheme for rating the reaction-to-fire performance of building products. Six 'Euroclasses' were established and test methods mandated for each. The Euroclasses comprise a poorly chosen



Privatisation of National Labs

- *“.....as these national labs become privatized/commercialized, the focus of their work will be more on income-producing activities, such as standardized testing, and less on true research.*
- *The net result has a dual negative effect: the further erosion of badly needed fundamental research programs and the spread of standardized testing of questionable value for certification”*

Croce, Interflam 2001

Fire Engineering - Summary

- Growing maturity of fire science and engineering coincided with world-wide trend towards deregulation
- Flexible, performance based design when done properly offers enormous benefits
- But there are areas that are not yet robust
- Research is essential for the safe implementation of performance based codes

Scientific
research
needs

"Research into,
and application of
Fire Safety
Engineering leads
to safe occupancy
of buildings
through cost
effective and
world class
design"



**A Research Strategy for the Fire Safety
Engineering design of buildings**



10 year research strategy

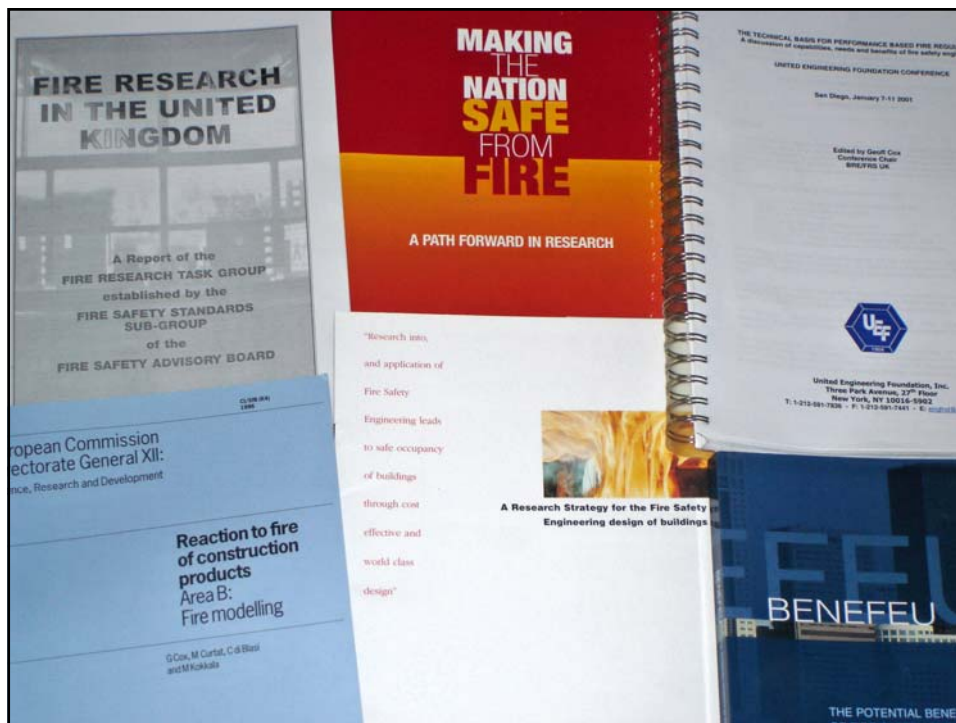
- *Department of the Environment (DoE) published “A Research Strategy for the Fire Safety Engineering Design of Buildings” 1995*
- *“.....achieve by the year 2005 a scientifically based and fully developed Fire Safety Engineering package (methodology furnished with appropriate calculation methods and data) that can be applied in a cost-effective way to the design and management of buildings.....”*
- WHERE IS IT????

Research for FSE

- Fire initiation & development
- Combustion products & smoke movement
- Passive protection
- Detection
- Active protection
- Evacuation
- Characteristic data
- Fire Statistics
- Risk assessment

Some topics

- Flame spread over solids
- under-ventilated fires
- Model development, RANS, LES, DNS models
- Turbulence/chemistry; Turbulence/radiation; boundary layers
- CFD model 'validation'/blind simulation challenges
- Application of advanced diagnostics for fires
- fire suppression 'science'
- development of 'smart' materials based on our knowledge of fire science



Research Requirements

- EC DG X11, Norbert Peters review on Reaction to Fire of Construction Products, 1996
- Intergovernmental Fire Research Group, 2001
- ODPM FSAB, 2003
- EU Benefeu, 2002
- US United Engineering Foundation, 2001
- US National Research Council, 2003
- SFPE “A Research Agenda for Fire Protection Engineering”, 2000

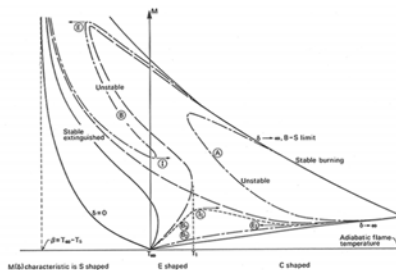


Fig. 2. Contours of constant ϕ (diagrammatic) for $\beta > 0$ and Lewis number unity.

THE SADDLE IN (M, T_a)

A first approximation to the position of the saddle point is obtainable from the indeterminacy of dT_a/dM which, from equation (8A), results in equations for the saddle values of M and T_a , from which

$$M_s \frac{\beta T_a}{T_s^2} \approx 0(1), \quad \frac{\beta T_a}{T_s^2} \gg 1,$$

and

$$\left(\frac{\alpha \delta e^{-T_a/T_s}}{3\beta} \right)_{\text{saddle}} \approx 1 + \frac{5}{2} \left(\frac{T_a}{T_s} - K \right)$$

$$\frac{\beta T_a}{T_s^2} \gg 1, \quad K > \frac{1}{6};$$

however, a detailed discussion of the position of the saddle necessitates a more precise statement regarding ϕ . Our computations for $\beta T_a/T_s^2 = 2$, for which Fig. 1 is a diagrammatic representation, confirm that there is then a saddle near to the

above values just within the region $T_a > T_{\infty}$. The saddle for high $\beta T_a/T_s^2$ moves into the region $T_a < T_{\infty}$.

If equation (6) is linearized and 'x' neglected in the denominator of unity, the resulting Airy's equation gives values of $M T_a \beta / T_s^2$ at the saddle ($T_a > T_s$) of about 1.56,

$$\frac{\alpha \delta e^{-T_a/T_s}}{\beta} \approx 2.5 \quad \text{and} \quad \frac{T_a}{T_s} (T_a - T_s) \approx 0.22.$$

within about 15% of our computations for $\beta T_a/T_s^2 = 50$. If the exponential approximation for the Arrhenius law is used in equation (6), it can be shown that $T_a/T_s \approx 2(T_a - T_{\infty})$ is, for $M \ll 1$, a function of $\alpha \delta M e^{-T_a/T_s} (T_a/T_s)$ and $M \beta T_a/T_s^2$ only. All these must take particular values at any saddle. Hence at the saddle $\alpha \delta e^{-T_a/T_s}$ is proportional to β for large $\beta T_a/T_s^2$.

THE C, S AND E CHARACTERISTICS

For large $T_a - T_{\infty}$ the $M(\beta)$ characteristic is C shaped, and for large negative $T_a - T_{\infty}$, $M(\beta)$ is the

Charles Babbage 1852

“Propose to any Englishman any principle or any instrument, however admirable, and you will observe that the whole effort of the English mind is directed to find a difficulty, a defect, or an impossibility in it.

If you speak to him of a machine for peeling a potato, he will pronounce it impossible; if you peel a potato with it before his eyes, he will declare it useless because it will not slice a pineapple.”

End